

GSFC · 2015

Continued Water-Based Phase Change Heat Exchanger Development

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Overview

- Why use Phase Change Material Heat Exchanger's (PCM HX's)?
- Prior PCM HX Development and Testing
- Full-Scale Design
- Prototypic Freezing and Thawing
- Unit A and Unit B Testing Summary
- Subscale Design and Testing Summary



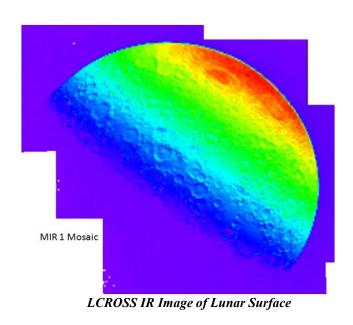
WHY USE A WATER-BASED PCM HX?

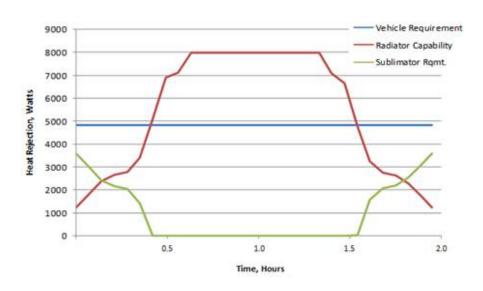


Why Use Water as a PCM?

- In cyclical heat load environments, a Supplemental Heat Rejection Device (SHReD) is required
 - Typically, accomplished through evaporators, sublimators, or Phase Change Material Heat Exchangers (PCM HX)
 - PCM's act a thermal battery and do not use a consumable
- Wax PCM is baseline for the Orion Spacecraft, but water is being investigated
 - Water has significantly higher latent heat of fusion than wax (333 kJ/kg vs. 163 kJ/kg)
 - Significant mass and volume savings possible

Problem: Water expands ~10% when frozen



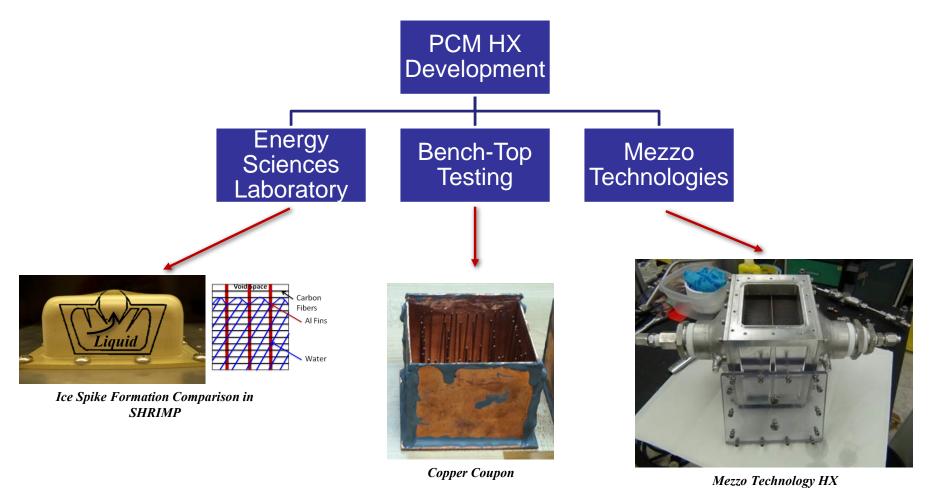




PRIOR PCM HX DEVELOPMENT AND TESTING



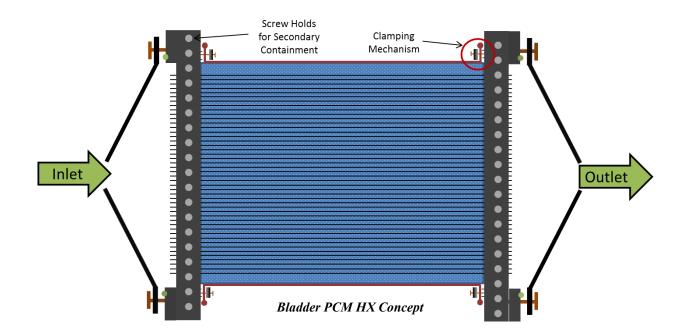
Why Use Water as a PCM?





Learning Outcomes from Historical Testing

- Providing extra void space may not necessarily lead to the development of a successful HX as void space will not necessarily be known in microgravity
- Use of "ice spike distribution" through inside-out freezing or uniform freezing should be pursued for future development
- Use of a bladder is feasible to implement with a water-based PCM HX



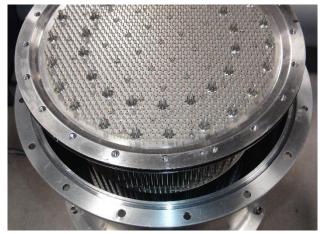


FULL-SCALE DESIGN AND DEVELOPMENT



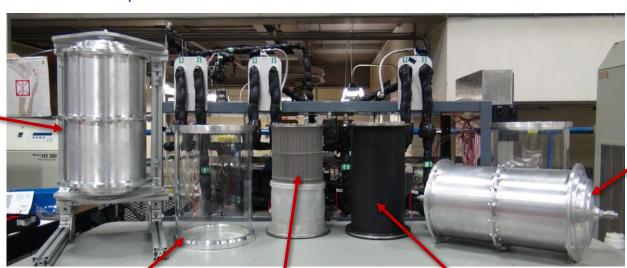
Full-Scale Design and Development

- 2 Orion-like full-scale PCM HX were designed and constructed through a SBIR Phase II with Mezzo Technologies
 - 3,700 kJ energy storage
- Each unit consists of 3 main components
 - Tube bank
 - Manifolds
 - Metal or "visualization" shell
- 12 kg of metal and 11.1 kg water
- Thermal analysis from Mezzo showed that it is possible to freeze/thaw in 90/30 minutes given sink temperatures and associated PCM inlet temperatures and flow rates



Tube Bank Being Installed into Shell

Metal Shell



Manifold

"Visualization" Shell

Tube bank

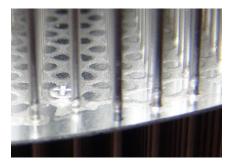
Tube bank with bladder installed

TFAWS 2015 – August 3-7, 2015 – Silver Spring, MD

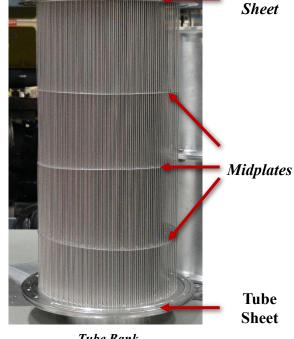


Full-Scale Design and Development (Cont.)

- Tube bank consists of 1,420 tubes
 - OD 0.042", ID0.035", wall thickness 0.0035"
 - 10% greater concentration in the center
 - Tubes brazed to tube sheets and midplates
- 2 Units constructed (EDU A & EDU B)
 - Units are essentially identically differing only by their midplate
 - Unit A: No holes in midplate
 - Unit B: Holes in midplate
- Viton GLT bladder manufactured through injection molding



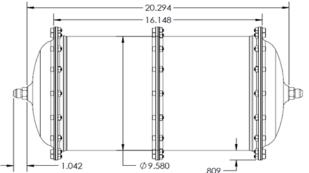
Unit A Midplate Design

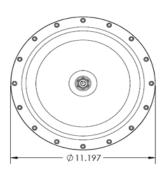


Tube Bank



Unit B Midplate Design





Tube

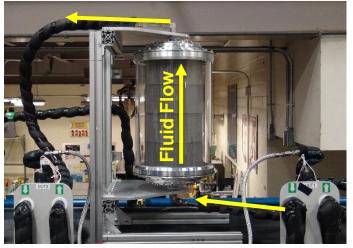


PROTOTYPIC FREEZING AND THAWING

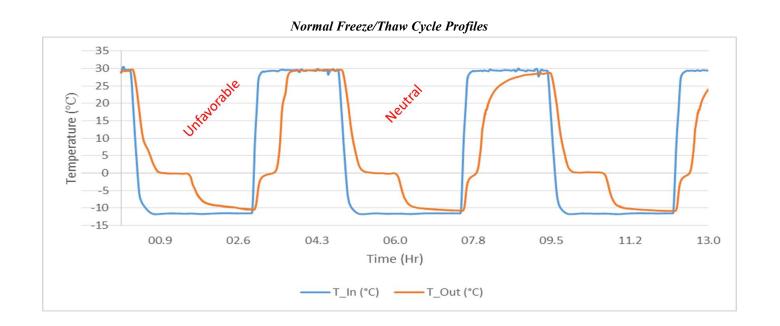


Test Set-Up

- Tested on RIP test stand at JSC
- 3 orientations tested
 - Favorable, unfavorable, neutral
- 50/50 PGW used as a working fluid with inlet and outlet temperatures of -12C to 30C controlled by chiller carts
- Transient testing also tested
- Bladder filled 100% with degassed water and 0% void space
- Goal was to achieve 100 cycles without failure



Favorable Testing Fluid Flow





Test Set-Up

- Bulge near outlet was expected to be observed during freezing but no bulge occurred
 - Due to bladder flexibility
 - Max expansion of ~1/2" (estimated at ¾")
 - Midplates did not have a noticeable effect on ice spike formation

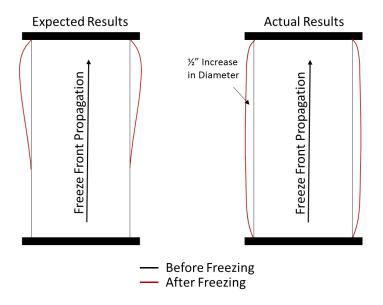




Image Comparison: Favorable



Image Comparison: Unfavorable

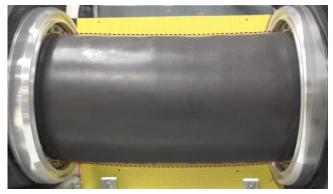
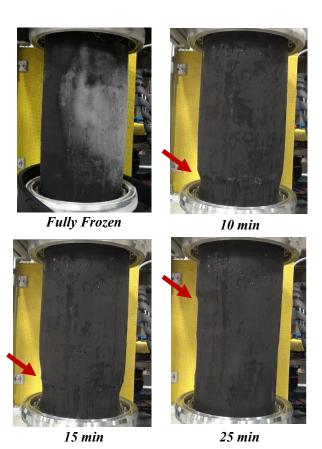


Image Comparison: Neutral



Prototypic Thaw Cycle

- Thawing did not occur as predicted
 - Hypothesized that bladder would simply return to its original position,
 - Bladder was pressed against tube bank because of decreased internal pressure due to melting water and corresponding decrease in volume
 - Caused significant bending in tubes around perimeter of the HX



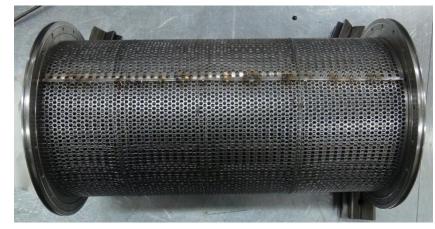


Unit A Bent Tubes Post-Test



Perforated Sheet Addition

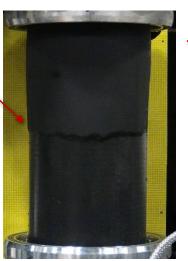
- To prevent tubes around the perimeter of the HX from bladder compression, a stainless steel perforated sheet was cut, rolled, and spot welded onto the tube bank core
- Testing resumed after sheet was in place



Perforated Sheet Installed









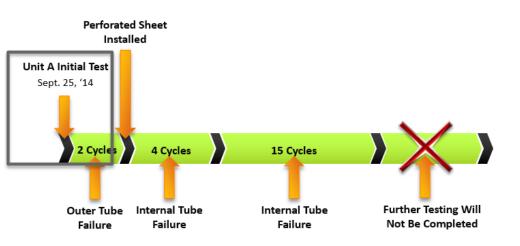


Thawing Progression with Perforated Sheet Installed

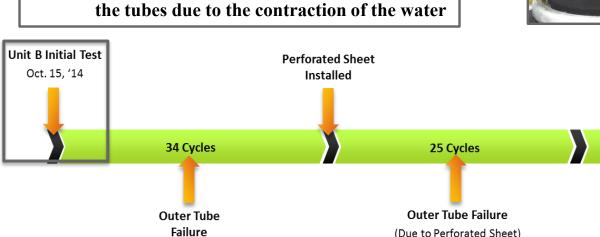


UNIT A AND UNIT B TESTING SUMMARY

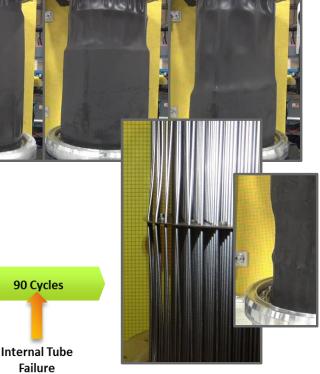




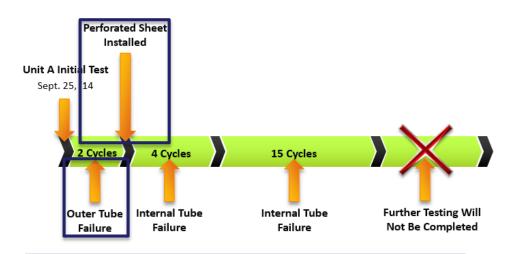
- Freeze and thaw tests were done on both EDUs with the Viton® bladder
 - Upon freezing the unit, the ice caused the bladder to expand as expected
 - Upon thawing, the bladder compressed onto the tubes due to the contraction of the water



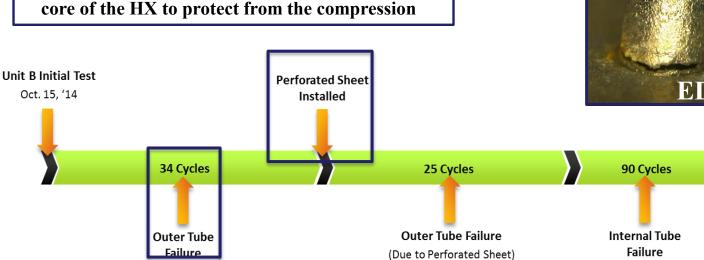




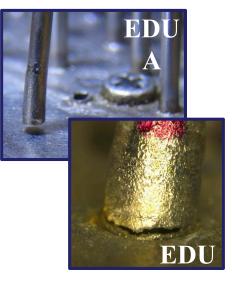




- Due to multiple compressions cycles of the bladder on to the perimeter tubes, a failure of an outside tube eventually occurred on both units
- A perforated sheet was wrapped around the tube core of the HX to protect from the compression

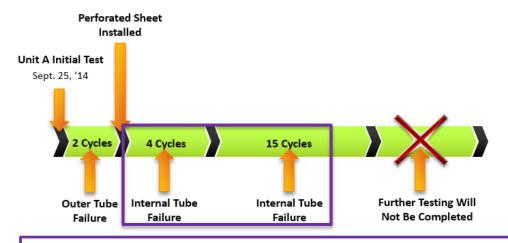




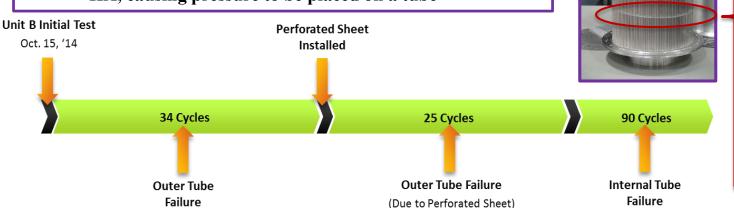


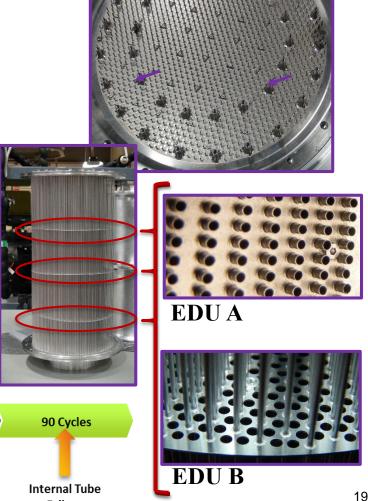




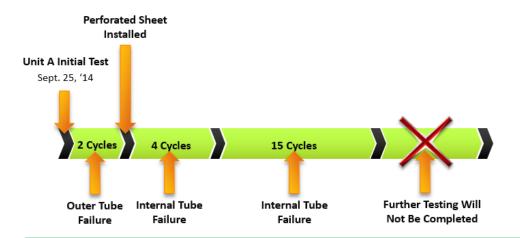


- Two internal tubes failed on EDU A
 - EDU A is the HX without perforated midplates
- The failure was hypothesized to be directly related to the freeze and thaw of the water on the inside of the HX
 - Water became hydraulically locked at midplate locations and did not have a path between various sections of the HX, causing pressure to be placed on a tube

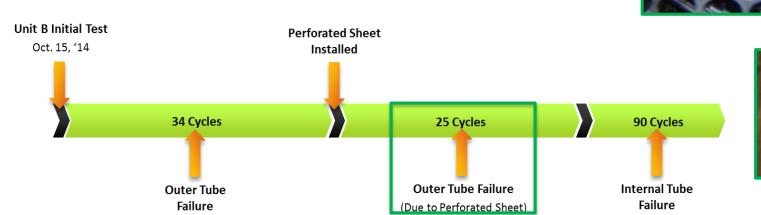




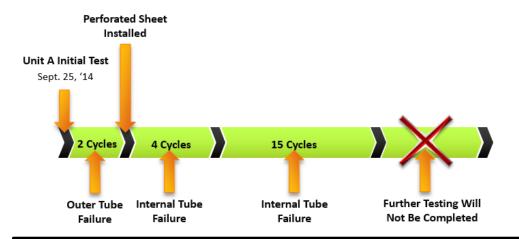




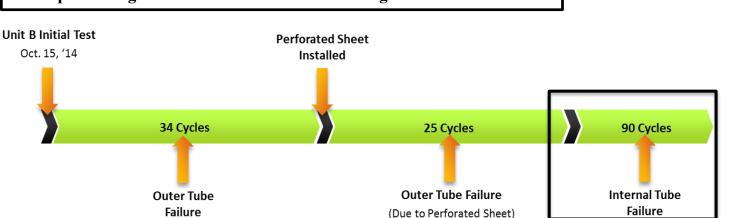
- During testing of Unit B, the bladder compressed onto the outer perforated sheet (meant to protect the outer tubes)
 - This caused the sheet to press against one of the tubes causing it to crack
- The tube was fixed and the perforated sheet was reinforced along the two edges to ensure this would not happen again

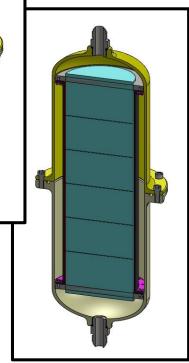






- Completed an additional 90 cycles on EDU B before an internal tube failed
- Location of failure could not be clearly identified during leak checking and tube sheet deformed during these tests so no further testing could be completed
- Continued with sub-scale PCM design and development using EDU B midplate design and lessons learned from testing EDU A and B





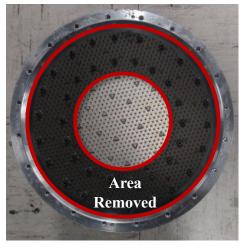


SUBSCALE DESIGN AND TESTING SUMMARY



Subscale Design

- Subscale PCM HX is a "cookie-cutter" of the full-scale unit
 - Approximately 4" in diameter and 10" long (full scale unit is 8.75" diameter and 16.1" long)
 - Designed to meet size requirements of ISS PCM HX Demonstration Loop and Orion pressure requirements
- Applied EDU A/B lessons learned to subscale unit including
 - Decreased aspect ratio
 - Use of perforated sheet and associated retaining ring
 - Use of more midplates
- Major differences between full-scale and subscale units include
 - Increased tube diameter
 - Epoxied not brazed to reduce uncertainty of braze process



"Cookie Cutter" of Full Scale Unit



<u>Characteristic</u>	<u>Full-Scale Unit</u>	Subscale Unit
Outer Diameter (in)	0.042	0.058
Tube Wall (in)	0.0035	0.0058
# of Tubes	1461	324

Subscale Unit Components

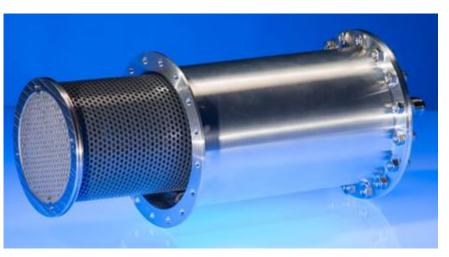


Subscale Testing Outcomes

- Testing with approximately the same flow rate per tube area as full scale unit
 - 85 lbs/hr for subscale unit verses 235 lbs/hr for full scale unit
 - Allows for scalability between full-scale and subscale units
- No major differences noticed between full-scale and subscale freeze/thaw cycling
 - Bladder expands evenly over length of HX and perforated sheet protects HX core during thawing
- 150 successful cycle completed as of July 14th!
 - 40 favorable, 21 neutral, 89 unfavorable cycles
- 1 minor testing issue
 - During cycle 19, the water fill screw developed a very minor leak which was observed in the temperature time data.
 - O-ring grease was added to threads and o-ring and testing resumed
 - No leaks or failures since leak was fixed



Tubes Inserted into Midplate



Subscale Core Inserted Into Shell



QUESTIONS?